



The University of Adelaide  
Department of Mechanical Engineering

**A STUDY OF POWER TRANSMISSION  
IN ACTIVELY CONTROLLED SIMPLE  
STRUCTURES**

Xia Pan

Submitted for the Degree of Doctor of Philosophy

August 1996

# Contents

Abstract	vi
Statement of originality	vii
Acknowledgments	viii
List of symbols	ix
<b>1 Overview</b>	<b>1</b>
1.1 Introduction	1
1.2 Literature review	2
1.2.1 Analysis of vibration in simple structures	2
1.2.2 Active control	3
1.2.3 Measurement of vibratory intensity in simple structures	8
1.3 New work	10
<b>2 Minimizing the forced response and vibratory power transmission in an infinite beam</b>	<b>11</b>
2.1 Introduction	11
2.2 Theory	12
2.2.1 Minimizing acceleration	12
2.2.2 Minimizing power transmission	13
2.3 Numerical results	16
2.3.1 Comparison of acceleration and power transmission control	16
2.3.2 Effect of error sensor and control force location	18

2.4	Comparison of theory with experiment . . . . .	26
2.5	Summary . . . . .	28
<b>3</b>	<b>Minimizing the forced response of a finite beam</b>	<b>29</b>
3.1	Introduction . . . . .	29
3.2	Theory . . . . .	30
3.2.1	Minimizing acceleration . . . . .	30
3.3	Numerical results . . . . .	33
3.3.1	Effect of boundary impedance . . . . .	33
3.3.2	Effect of control force location . . . . .	37
3.3.3	Effect of error sensor location . . . . .	39
3.3.4	Effect of forcing frequency . . . . .	43
3.3.5	Comparison with control of semi-infinite beam . . . . .	46
3.4	Summary . . . . .	51
<b>4</b>	<b>Minimizing acceleration and power transmission in a semi-infinite plate</b>	<b>52</b>
4.1	Introduction . . . . .	52
4.2	Theory . . . . .	53
4.2.1	Minimization of acceleration with a line of in-phase control forces . . . . .	53
4.2.2	Minimization of acceleration with a line of three independently driven control forces . . . . .	54
4.2.3	Power transmission . . . . .	55
4.2.4	Minimization of power transmission with a line of in-phase point control forces . . . . .	56
4.2.5	Minimization of power transmission with a line of three independently driven point control forces . . . . .	58
4.3	Numerical results . . . . .	59
4.3.1	One primary and one control force . . . . .	60
4.3.2	A row of in-phase, uniform amplitude control forces and a single primary force . . . . .	67
4.3.3	A row of in-phase, uniform amplitude control forces and a row of in-phase, uniform amplitude primary forces . . . . .	69

4.3.4	Effect of forcing frequency on performance with in-phase control forces	71
4.3.5	A row of three in-phase primary forces and a row of three independently driven control forces . . . . .	72
4.3.6	Effect of error sensor type and location . . . . .	76
4.4	Summary . . . . .	81
<b>5</b>	<b>An experimental study of active control of power transmission characteristics in a semi-infinite plate</b>	<b>82</b>
5.1	Introduction . . . . .	82
5.2	Experimental arrangement . . . . .	83
5.3	Test procedure . . . . .	84
5.4	Numerical and experimental results . . . . .	86
5.4.1	Active control of power transmission . . . . .	91
5.4.2	Intensity vortices . . . . .	93
5.5	Discussion and summary . . . . .	98
<b>6</b>	<b>Piezoelectric actuator vs point force excitation of a beam and a plate</b>	<b>100</b>
6.1	Introduction . . . . .	100
6.2	Response of infinite beam excited by a pair of piezoelectric actuators . . . . .	101
6.3	Response of semi-infinite plate driven by a pair of piezoelectric actuators . . . . .	105
6.4	Power transmission along a semi-infinite plate excited by a pair of piezoelectric actuators . . . . .	109
6.5	Discussion and summary . . . . .	111
<b>7</b>	<b>Minimizing acceleration and power transmission in a semi-infinite cylinder</b>	<b>113</b>
7.1	Introduction . . . . .	113
7.2	Theory . . . . .	114
7.2.1	Minimization of acceleration with a line of in-phase control forces . . . . .	114
7.2.2	Minimization of acceleration with a line of three independently driven control forces . . . . .	116
7.2.3	Power transmission . . . . .	117

7.2.4	Minimization of power transmission with a line of in-phase point control forces . . . . .	118
7.2.5	Minimization of power transmission with a line of three independently driven point control forces . . . . .	120
7.3	Numerical results . . . . .	122
7.3.1	Definition of the near field . . . . .	122
7.3.2	Power transmission reduction . . . . .	123
7.3.3	Effect of error sensor type, location and number . . . . .	125
7.3.4	Effect of control force type and location . . . . .	132
7.3.5	Effect of thickness, radius and frequency . . . . .	134
7.4	Summary . . . . .	136
<b>8</b>	<b>An experimental study of active control of power transmission in a semi-infinite cylinder</b>	<b>137</b>
8.1	Introduction . . . . .	137
8.2	Experimental arrangement . . . . .	137
8.3	Test procedure . . . . .	142
8.4	Numerical and experimental results . . . . .	143
8.5	Summary . . . . .	147
<b>9</b>	<b>Conclusions and recommendations</b>	<b>149</b>
<b>A</b>	<b>Classification of beam boundary conditions</b>	<b>152</b>
A.1	Beam boundary impedance . . . . .	152
A.2	Equivalent boundary impedance of an infinite beam . . . . .	157
<b>B</b>	<b>Response of a finite beam to a point force</b>	<b>159</b>
<b>C</b>	<b>Response of a semi-infinite plate to a line of point forces driven in phase</b>	<b>162</b>
<b>D</b>	<b>Modal decomposition method in a semi-infinite plate</b>	<b>167</b>
<b>E</b>	<b>Measurement of amplitude reflection coefficient in a semi-infinite plate</b>	<b>169</b>

<b>F</b>	<b>Response of a semi-infinite cylinder to a line of point forces driven in phase</b>	<b>171</b>
F.1	Determining the wavenumbers and constants $\alpha$ and $\beta$ . . . . .	172
F.2	Determining the flexural wavelength . . . . .	175
F.3	Simply supported end conditions . . . . .	175
F.4	Equilibrium conditions at the point of an applied force . . . . .	176
F.5	Determination of the eigenvector . . . . .	177
<b>G</b>	<b>Modal decomposition method in a semi-infinite cylinder</b>	<b>180</b>
<b>H</b>	<b>Measurement of vibratory intensity in simple structures</b>	<b>181</b>
H.1	Measurement of vibratory intensity in an infinite beam . . . . .	181
H.2	Measurement of vibratory intensity in a semi-infinite plate . . . . .	182
H.3	Measurement of vibratory intensity in a semi-infinite cylinder . . . . .	184
	<b>References</b>	<b>189</b>
	<b>Publications originating from this thesis work</b>	<b>196</b>

# Abstract

Feedforward active control of harmonic vibratory power transmission in simple structures is investigated theoretically and experimentally. The structures investigated are a beam, a plate and a cylinder. Primary excitation is used to represent unwanted noise or vibration. Secondary excitation is introduced using control sources which are adjusted to minimize acceleration or power transmission in the structures. The primary and secondary excitation is produced by either electromagnetic force actuators (shakers) or piezoelectric ceramics.

The theoretical predictions are compared to the experimental test results. In addition, vibratory intensity distributions before and after control are investigated.

Both the theoretical and experimental results demonstrate that it is possible to minimize vibratory power transmission in the test simple structures using a maximum of three control sources in the test frequency range. The study also indicates that, in most cases, the harmonic vibratory power transmission in simple structures can be measured by using a maximum of two accelerometers in the test frequency range.